

Interrogative Simulation and Uncertainty Quantification of Multi-Disciplinary Systems

Ali H. Nayfeh and Muhammad R. Hajj Department of Engineering Science and Mechanics Virginia Polytechnic Institute and State University Blacksburg, VA 24061

Bifurcation and Model Reduction Techniques for Large Multi-Disciplinary Systems University of Liverpool 26-27 June, 2008

Background



- Simulation, testing and prediction of the responses of multidisciplinary systems are extremely difficult
 - Nonlinear competing mechanisms (e.g. aeroelastic phenomena)
 - Multiple coexisting attractors, supercritical and subcritical bifurcations, limit cycles, etc.

Global Metrics

- Flutter speed, LCO amplitude
- Lift and drag forces on bluff bodies
- Pull-in voltage and displacement of MEMS
- Dependence on physical parameters is not straightforward.

Motivation



- Difficult and expensive to use brute- force computations to relate global response metrics to system parameters
- Need for reduced-order models
 - Reproduce results of high-fidelity simulations or experiments
 - Relate global metrics to system parameters
 - Quantify system response uncertainties
 - Implement control strategies (flow control)
- Physically motivated models
 - Make use of scientific principles, feasible for certain systems (simple structures, ...)
- Black-box models
 - No clear relation between physics and system (networks, ...)
 - Match the behavior with that of a mathematical model

Objective



Interrogative Testing/Simulation and Uncertainty Quantification of Multi-Disciplinary Systems

- Approach Design the experiment/simulation to
- Exploit specific behavior
- Draw more information from data
- Relate model features to system dynamics
- Examples
- System identification
- Global metrics
- Uncertainty quantification
- Control (flow control)

System Identification HSCT – FSM



A. H. Nayfeh & M. R. Hajj

Virginia

Tech

System Identification HSCT – FSM





Response is different in subsonic, transonic and supersonic flow regimes.

Exploitation of specific behavior in different regimes

June 2008

Exploitation of Nonlinearities

Nonlinear Response



Structure Vibrations, Ship Motions, Lift and Drag on Cylinders

Virginia

èch

System Identification F-15 tail Assembly Model



Model

- 1/16 dynamically scaled model
- 0.355 *m* x 0.28 *m* x 0.482 *m*
- Series of aluminum channels, brass rings, composite plates, metal masses,...
- Excitation
 - Mounted on a 250-lb shaker
 - Exploit parametric resonance to maximize influence of nonlinearities
- Identification Procedure
 - Series of experiments
 - Combination of approximate solutions for governing equations and data analysis
- Control and effects of uncertainties



System Identification Governing Equations

 u_1, u_2 : modal coordinates (micro strains (µs)) ω_1, ω_2 : natural frequencies (*radian / s*) 1, 2 : linear damping coefficients (*radian / s*) 3, 4 : aerodynamic damping coefficients.($1/\mu s$) 1, 2 : coefficients of cubic nonlinearity ($1/\mu s^2 * s^2$) 1, 2 : transmissibility terms ($1/g * s^2$)

k : coupling term.
$$(1/s^2)$$



Virginia

$$\ddot{u}_{1} + \omega_{1}^{2} u_{1} + 2\mu_{1}\dot{u}_{1} + \alpha_{1}u_{1}^{3} + \mu_{3}\dot{u}_{1}|\dot{u}_{1}| - k \Psi_{2} - u_{1} = u_{1} \eta_{1} F \cos \Omega t + \tau_{1}$$
$$\ddot{u}_{2} + \omega_{2}^{2} u_{2} + 2\mu_{2}\dot{u}_{2} + \alpha_{2}u_{2}^{3} + \mu_{4}\dot{u}_{2}|\dot{u}_{2}| - k \Psi_{1} - u_{2} = u_{2} \eta_{2} F \cos \Omega t + \tau_{2}$$

Linear Identification: Frequency Response Functions

 f_1 , f_2 : 9.135 Hz, 9.05 Hz damping ratios: 0.014 and 0.019 - will also be identified from parametric excitation experiments

A. H. Nayfeh & M. R. Hajj

System Identification Nonlinear Identification

$$\ddot{u}_{1} + \omega_{1}^{2} u_{1} + 2\mu_{1}\dot{u}_{1} + \alpha_{1}u_{1}^{3} + \mu_{3}\dot{u}_{1}|\dot{u}_{1}| - k \Psi_{2} - u_{1} = u_{1} \eta_{1} F \cos Q t + \tau_{1}$$
$$\ddot{u}_{2} + \omega_{2}^{2} u_{2} + 2\mu_{2}\dot{u}_{2} + \alpha_{2}u_{2}^{3} + \mu_{4}\dot{u}_{2}|\dot{u}_{2}| - k \Psi_{1} - u_{2} = u_{2} \eta_{2} F \cos Q t + \tau_{2}$$

Uncoupled experiments – one tail fixed (*k=0*); $\Omega = 2\omega_1 + \varepsilon \sigma_1$

Multiple scales $u_1 = A_1(\varepsilon t)e^{i\omega_1 t} + \varepsilon(\dots) + \dots + cc$ $A_1 = \frac{1}{2}ae^{i\beta}$

$$a' = -\mu_{1}a - \frac{4\mu_{3}}{3\pi}\omega_{1}a^{2} + \frac{\eta_{1}Fa}{4\omega_{1}}\sin\gamma$$

$$\varsigma_{1} = 0.01357, \quad \mu_{3} = 3.157 \times 10^{-4}\mu\varepsilon^{-1},$$

$$a\beta' = \frac{3\alpha_{1}}{8\omega_{1}}a^{3} - \frac{\eta_{1}Fa}{4\omega_{1}}\cos\gamma$$

$$\alpha_{1} = -3.675 \times 10^{-2}\frac{1}{s^{2}\mu\varepsilon^{2}}, \quad \eta_{1} = 161.54\frac{1}{gs^{2}}$$

$$\gamma = \sigma_{1}T_{1} - 2\beta + \tau_{1}$$

$$\varsigma_{2} = 0.01856, \quad \mu_{4} = 1.9588 \times 10^{-4}\mu\varepsilon^{-1},$$

$$c_{1} = -3.675 \times 10^{-2}\frac{1}{s^{2}\mu\varepsilon^{2}}, \quad \eta_{1} = 161.54\frac{1}{gs^{2}}$$

Steady state solution a' = 0 and $\gamma' = 0$

 $\alpha_2 = -2.977 \times 10^{-3} \frac{1}{s^2 \mu \varepsilon^2}, \ \eta_2 = 275.12 \frac{1}{g s^2}$

Virginia

A. H. Nayfeh & M. R. Hajj

System Identification Nonlinear Identification

$$\ddot{u}_{1} + \omega_{1}^{2} u_{1} + 2\mu_{1}\dot{u}_{1} + \alpha_{1}u_{1}^{3} + \mu_{3}\dot{u}_{1}|\dot{u}_{1}| - k \Psi_{2} - u_{1} = u_{1} \eta_{1} F \cos Q t + \tau_{1}$$
$$\ddot{u}_{2} + \omega_{2}^{2} u_{2} + 2\mu_{2}\dot{u}_{2} + \alpha_{2}u_{2}^{3} + \mu_{4}\dot{u}_{2}|\dot{u}_{2}| - k \Psi_{1} - u_{2} = u_{2} \eta_{2} F \cos Q t + \tau_{2}$$



A. H. Nayfeh & M. R. Hajj

Virginia

Tech

Three-beam Frame

$$\ddot{u} + \omega^{2}u + 2\varepsilon^{2}\mu_{1}\dot{u} + \varepsilon^{2}\mu_{2}\dot{u} | \dot{u} |$$

$$\varepsilon\alpha_{2}u^{2} + \varepsilon^{2}\alpha_{3}u^{3} + \varepsilon^{2}\delta\dot{u}^{2}u =$$

$$\varepsilon\eta_{1}f\cos(\Omega t + \tau_{e}) + \varepsilon^{2}\eta_{2}fu\cos(\Omega t + \tau_{e})$$





Virginia

Tech



- Data Source: Large-Amplitude-Motions Program (LAMP)
- □ Nonlinear Model: 3 DOF: heave, pitch, roll
- Approximate Solution: Method of Multiple Scales
- Higher-order spectral analysis: spectral parameters and in particular phase measurement
- Account for multiple phase quantities

Roll Instabilities in Ship Motions



Virginia

ech

Lift on Stationary Circular Cylinders



Data Source: Numerical simulation



Approximate Solution: Method of Multiple Scales

Higher-order spectral analysis (trispectrum): spectral parameters and in particular phase measurement

Lift on Stationary Circular Cylinders



Lift Modeling and Approximate Solution

Rayleigh Equation

$$\ddot{l} + \omega_s^2 l - \mu_r \dot{l} + \alpha_r \dot{l}^3 = 0$$

$$l = a \cos(\omega_s t + \beta) + \frac{\alpha_r \omega_s}{32} a^3 \cos(3\omega_s t + 3\beta - \frac{1}{2}\pi)$$

$$\phi(3\omega_s) - 3\phi(\omega_s) = -\pi/2$$

van der Pol Equation

 $\ddot{l} + \omega_s^2 l - \mu_v \dot{l} + \alpha_v l^2 \dot{l} = 0$

$$l = a\cos(\omega_s t + \beta) + \frac{\alpha_v}{32\omega_s}a^3\cos(3\omega_s t + 3\beta + \frac{1}{2}\pi)$$

$$\phi(3\omega_s) - 3\phi(\omega_s) = \pi/2$$

F16 flight testing

Relate global metrics to system parameters



A. H. Nayfeh & M. R. Hajj

Virginia

ech

Goland wing with Store Virginia Flutter Speed





The <u>complex conjugate</u> representing the second mode cross the imaginary axis transversely, and the instability is due to a <u>Hopf bifurcation</u>.



ech

Goland wing with Store

Flutter speed variations with uncertainties in damping

$$\begin{split} M\ddot{q}(t) + \frac{UL}{\tau}(D+k_1D_1+k_2D_2)\dot{q}(t) + (K_s+K_eU^2)q &= 0 \\ U &= U_f + \epsilon \ U_1 \\ q(t;\epsilon) &= q_1(T_0,T_1) + \epsilon \ q_2(T_0,T_1) + \cdots \\ q_1 &= A(T_1)v_f e^{i\omega_f T_0} + \bar{A}(T_1)\bar{v}_f e^{-i\omega_f T_0} \\ A(T_1) &= \frac{1}{2}r(T_1)e^{i\beta(T_1)} \\ \frac{d}{dt} r &= [0.00190782(U-U_f) - 0.00218554k_1 + 0.00218554k_2] \ r \\ \frac{d\beta}{dt} = [0.0376153(U-U_f) - 0.00104746k_1 + 0.0013755k_2] \end{split}$$

Variations in flutter speed due to uncertainties in the damping parameters k_1 and k_2 $U_1 = U - U_f = 1.14558k_1 - 0.345961k_2$

June 2008

A. H. Nayfeh & M. R. Hajj

Virginia

 $k_2 = 0$

390

380

370

360

350

340

330

0

0.2

0.4

èch

Goland wing with Store

Nonlinear parameter uncertainty



Variations in the flutter speed due to uncertainties in other parameters

$$U_1 = U - U_f = \Gamma_1 k_1 - \Gamma_2 k_2 + \Gamma_3 \delta M_s + \Gamma_4 \delta D_x + \Gamma_5 \delta D_y$$

Parameters Uncertainty and Effects on Global Measures – Nonlinear analysis $M\ddot{q}(t) + \frac{UL}{\tau}(D + k_1D_1 + k_2D_2)\dot{q}(t) + (K_s + K_eU^2)q = N_s(q) + N_{store}(q) + N_{aero}(q)$

$$\begin{aligned} v_f^*(2i\omega_f M + \frac{U_f L}{\tau}D)v_f A' + v_f^* \left(\frac{i\omega_f L}{\tau}D + 2U_f K_e\right)v_f U_1 A \\ &\quad + \frac{i\omega_f U_f L}{\tau}v_f^*(k_1 D_1 + k_2 D_2)v_f A = (\Lambda_s + \Lambda_{store} + \Lambda_{aero})A^2 \bar{A} \\ &\quad \frac{d}{d} \frac{r}{t} = -\left[\kappa_1(U - U_f) + \gamma_{11}k_1 + \gamma_{12}k_2\right]r - \alpha r^3 \\ &\quad \frac{d}{d} \frac{\beta}{t} = -\left[\kappa_2(U - U_f) + \gamma_{21}k_1 + \gamma_{22}k_2\right] + \chi r^2 \end{aligned}$$

A. H. Nayfeh & M. R. Hajj

ROM of the Velocity Field

Virginia Tech

Project Navier-Stokes equation onto the POD modes

ROM (*M*=10, Re=100)

- Ordinary-differential equations
- Nonlinear system

Linear stability – eigenvalues

- Pair in right-half plane
- Hopf bifurcation

Actuators Placement





Control of vortex shedding Virginia



Summary



- System characterization and ROM development through exploitation of physical behavior
- Flight testing: Combination of understanding of physical behavior and data analysis to develop a model with parameters that depend on system variables and system characterization

Goland wing + store

- Modeling stages
- Uncertainty quantification framework
- Use of ROM derived physical characteristics to control vortex shedding